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E-25-651

GEORGIA INSTITUTE OF TECHNOLOGY
School of Mechanical Engineering
Atlanta, Georgia



PROGRESS REPORT
PHASE I

LARGE GAS TURBINE WHEELSPACE COOLING STUDIES

G. E. SERVICE AGREEMENT
P. O. #087-ETEL-71225

by

Ward O. Winer
Professor
and
Principal Investigator

David M. Sanborn
Associate Professor

Scott Bair
Research Engineer

Sponsored by
General Electric Corporation
Large Gas Turbine Division
Schenectady, New York 12345

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School of Mechanical Engineering
Atlanta, Georgia

PROGRESS REPORT
PHASE I
SUMMARY

LARGE GAS TURBINE WHEELSPACE COOLING STUDIES

This report describes a facility to simulate turbine wheelspace cooling characteristics of General Electric medium and large gas turbines and contains some preliminary data obtained. The system is capable of operating with hot rim flow which is turned in a manner simulating actual machine behavior. It has been found that hot gas inflow to the wheelspace results in part from circumferential variations in the rim flow and persists even to relatively high net outward cooling flow passed the seals.

Ward O. Winer
Principal Investigator

Stothe P. Kezios, Director
School of Mechanical Engineering

October, 1976

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I. INTRODUCTION

The objective of the efforts reported here was to develop a facility to simulate wheelspace cooling characteristics of General Electric medium and large gas turbines, and to obtain preliminary data with the facility. The wheelspace cooling development program of which this work is a part is summarized in a General Electric Memorandum by R. F. Hoeft dated October 1, 1974. The historical perspective of the program is presented in that memorandum and the references listed in it.

The objectives of the wheelspace cooling program are:

1. To provide a working curve for establishing cooling flow requirements on actual designs.
2. To evaluate different seal and wheelspace configurations that have not been previously evaluated.
3. To continue efforts towards understanding the fluid dynamics of this system to allow predicting the effect of parameters not anticipated in this program.

The work reported here is the first phase of a continuing effort to meet the above objectives.

The facility developed consists of a 40 inch diameter wheel without buckets in a casing which has stationary blades to simulate nozzle flow and the turning of flow over the wheel rim. Heated mainflow is provided which simulates actual turbine mainflow velocity in both magnitude and direction. Cooling flow to both forward and aft wheelspaces can be varied. Both forward and aft stator-wheel spacing can be varied. Wheel speed can be varied continuously up to 3000 RPM. Several interchangeable wheel and stator seal pieces are available.

The instrumentation consists of flow measurements of forward

and aft cooling flows, and main flow as well as 54 thermocouples and 44 static pressure taps.

The preliminary test program reported consisted of a series of experiments on a single seal configuration with varying wheel speed and cooling flows. The primary result was the observation that circumferential variations in the mainflow cause circumferential variations in seal flow which could not be adequately determined with the original pressure and temperature sensors. Increased instrumentation density circumferentially in the facility verified this observation. The source of the circumferential variation in the mainflow is the mainflow inlet port distribution on the apparatus.

II. THE EXPERIMENTAL FACILITY

The test apparatus consisting of the rotor, the housing, and seals were supplied by General Electric. Instrumentation, drive train and plumbing were supplied by Georgia Tech.

A. Wheelspace Apparatus:

The wheelspace apparatus is shown in five photographs in Appendix B and the five mechanical drawings for it are reproduced as foldouts in Appendix C.

A 40 inch diameter, balanced rotor with fore and aft rotating seals was mounted on permanently lubricated rolling element bearings in the walls of the housing. No rotating buckets were attached to the wheel. Simulated crossflow was provided through fifteen discrete openings to an annular flow space surrounding the rotor. Stationary buckets at the axial position of the wheel, and nozzles upstream of the wheel provided for a flow direction at the rim openings comparable to operating turbine wheels. A plenum was included upstream of the nozzles to smooth the stagnation pressure distribution arising from the crossflow entrances. Crossflow was exhausted directly to the atmosphere from ten openings in the housing. Nozzles and buckets were mounted in a half-ring which may be removed for inspection of rim and seal spacings.

Axial rotor-to-rim spacing was adjustable through jack bolts in the housing wall and was independent of inner wall to rotor spacing (also adjusted with jack bolts). Stationary seals were mounted to the fore and aft inner walls to interact with the rotating seals with a clearance regulated by shims and set screws. Seal overlap was changed by employing stationary seals of varying width.

Cooling through-flow was introduced to the wheelspace through three openings in each the fore and aft sides. No adjustment of the wheelspace wall to rotor spacing was provided.

The rotor shaft was attached through a Lovejoy flexible coupling to a drive shaft running beneath a propane fueled Chrysler industrial engine fitted with an electronic speed limiter. The engine and drive-shaft were connected by a belt and a hand operated dry clutch. The drive shaft was supported by two pillow blocks. The belt drive also acted as a torque limiting device. Rotor speed was measured by a mechanical tachometer held against the rotor shaft or from calculation using belt reduction ratio and engine RPM measured by an electronic tachometer on the engine ignition.

Crossflow was supplied by a Worthington two-stage piston air compressor at four pounds per sec. The air passed through a combustion chamber before entering the distribution hoses to the rotor housing. An air-propane mixture was electrically ignited in the combustion chamber to provide an exit temperature of 250F. Combustion was controlled pneumatically by a Taylor Instruments Temperature Controller. A flame-out sensor and low air/propane pressure switches were provided for safety.

Air flow rate to the burner was measured with an orifice plate flowmeter upstream of the runner and air flow control valve which was opened full for all tests. Pressure was measured at the same location. Fifteen reinforced rubber hoses provided distribution of cross flow to the housing.

Cooling through-flow was provided at up to 1.3 pounds per sec from a 10 hp electric centrifugal blower through three ducts on the fore

and aft sides of the housing. The rate of cooling air flow was measured with orifice plate flowmeters in each of the two pipes leading to the ducts. In this way fore and aft flows were measured separately. Total cooling air flow was regulated by restricting the inlet of the blower. A thermocouple in one inlet pipe was used to indicate cooling inlet temperature.

B. Instrumentation:

Housing instrumentation consisted of 44 static pressure taps and 54 copper-constantan thermocouples concentrated at three circumferential positions. The instrumentation was distributed radially in three positions along the wheelspace wall and radial seal area, and axially along the crossflow space. In addition three radial positions were included for wheelspace thermocouples both fore and aft. The instrumentation sensor locations are shown in Figure 1 and 2.

The static pressures were read on a common-well manometer with reference to atmospheric pressure. The temperatures were obtained by connecting the thermocouples to a Leeds and Northrup multipoint recorder. Table I indicates the relationship between recorder position and thermocouple location.

Twelve of the pressure taps were only installed for the last two tests - M and N. These were placed in pairs circumferentially on the outer cross flow surface to detect circumferential variations in the cross flow.

Table I. Thermocouple Identification - December 15, 1975

Recorder Bank	T/C	MACHINE LOCATION	Bank	T/C	LOCATION
1	2	E73	4	1	C11
	2	D71		2	C31
	4	C01		3	B61
	5	C74		5	B63
	6	C71		6	B64
	7	C84		7	B51
	8	C81		8	B53
	9	E81		9	B41
	10	E83		10	B43
	11	D81		11	B44
	12	F71		12	B31
2	1	F73	5	1	B33
	3	A71		2	B21
	4	B71		3	B23
	5	B74		4	B24
	6	F81		6	B11
	7	B81		7	A01
	8	F83		8	D01
	9	B84		9	B03
	10	A81		10	C03
	11	E71		11	B01
	12	C44		12	
3	1	C64	6	1	B04
	2	C63		2	C04
	4	C61		3	
	5	C53		4	
	6	C51		5	
	7	C43		6	
	8	C41		7	
	9	C33		8	
	10	C24		9	
	11	C23		10	AMBIENT
	12	C21		11	CROSS INLET
				12	COOLING INLET 1/16/76

56 THERMOCOUPLES
54 PRESSURE TAPS

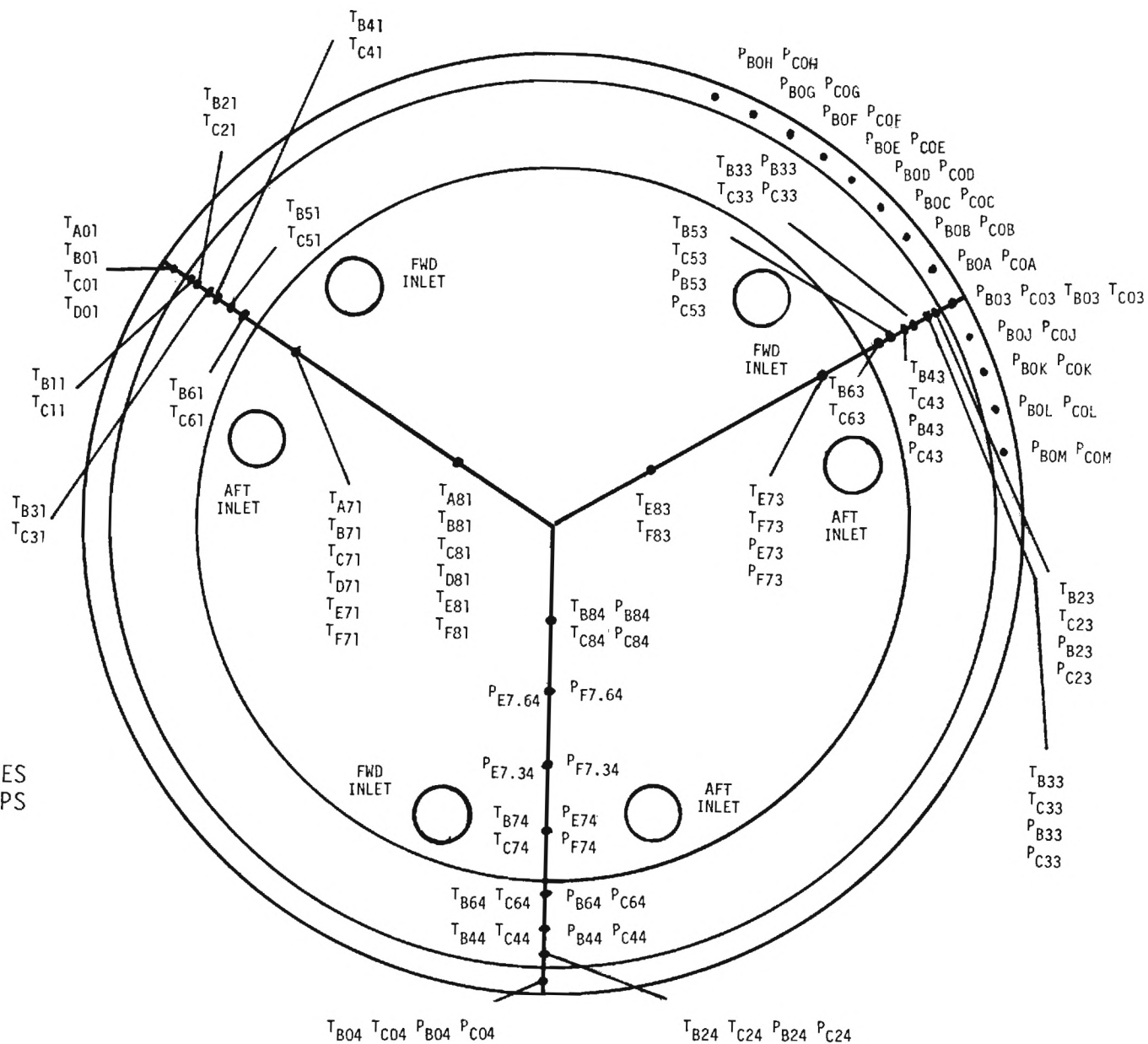


Figure 1. Instrumentation (Viewed from Aft End)

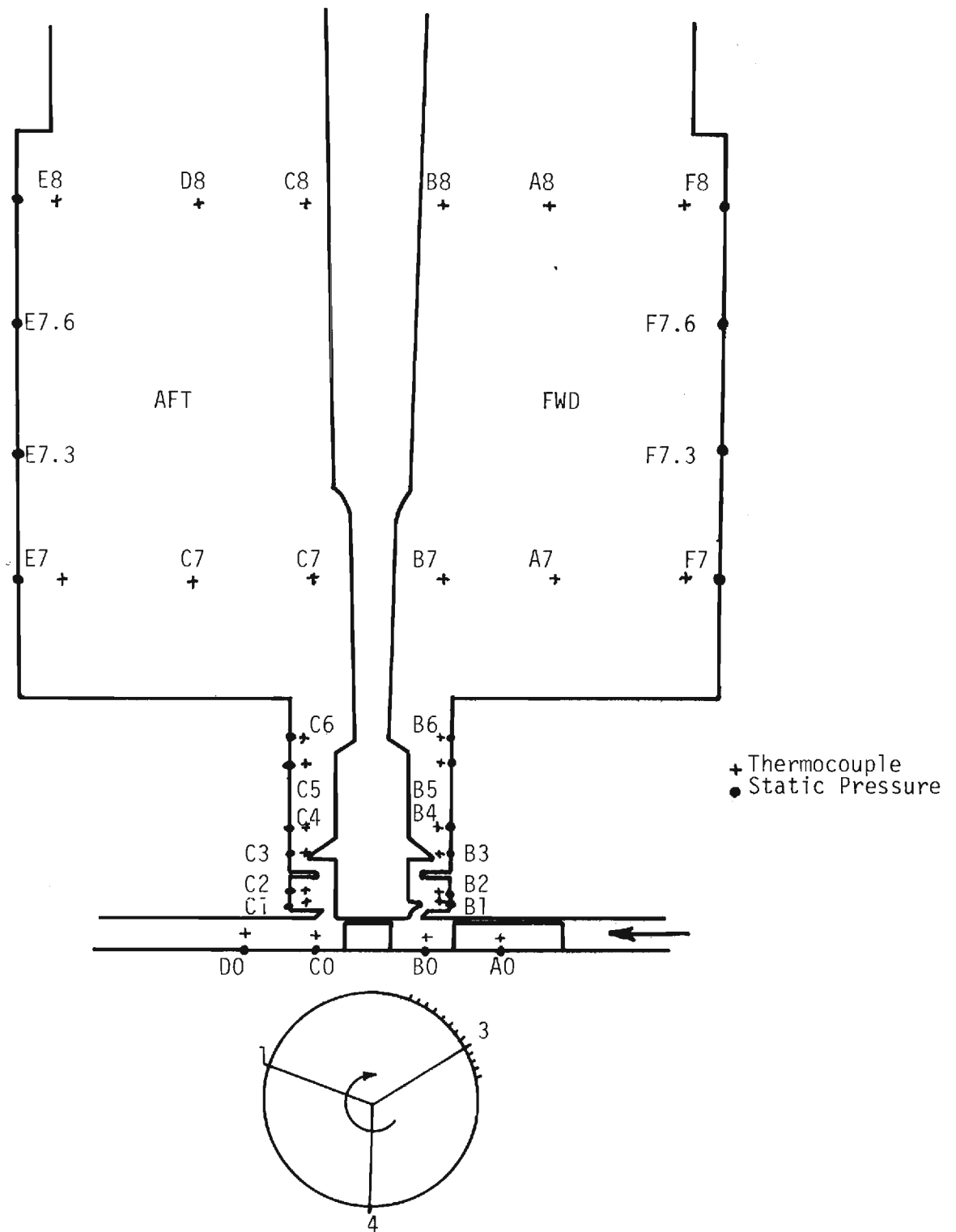


Figure 2. Instrumentation.

III. TEST PROCEDURE

Only steady state data were of interest in this study. The procedure was to start the compressor and burner for the cross flow, the blower for the cooling flow and engine for wheel rotation. When hot cross flow was employed the thermal transient of the system was the controlling factor to determine when steady state was reached. The temperatures as read on the multipoint recorder were used to determine steady state. When cold cross flow was used the wheel speed and pressures as indicated by the heights in the manometers were used to determine steady state. In this case steady state was reached in one to three minutes. Once steady state was reached the manometer readings and flowrates were recorded, and the temperatures automatically recorded.

In the set of tests reported the crossflow (4.0 lbm/s) and seal geometry were held constant. The seal geometry used is shown in Figure 3. The parameters varied were wheel speed, cross flow temperature, cooling flow rate and cooling flow entrance ports. Table II lists a summary of test conditions and readings taken. In all cases except test N the cooling flow fore and aft were measured - in run N there was no forced cooling flow but the parts were not blocked. In the cold cross flow tests temperatures were not recorded.

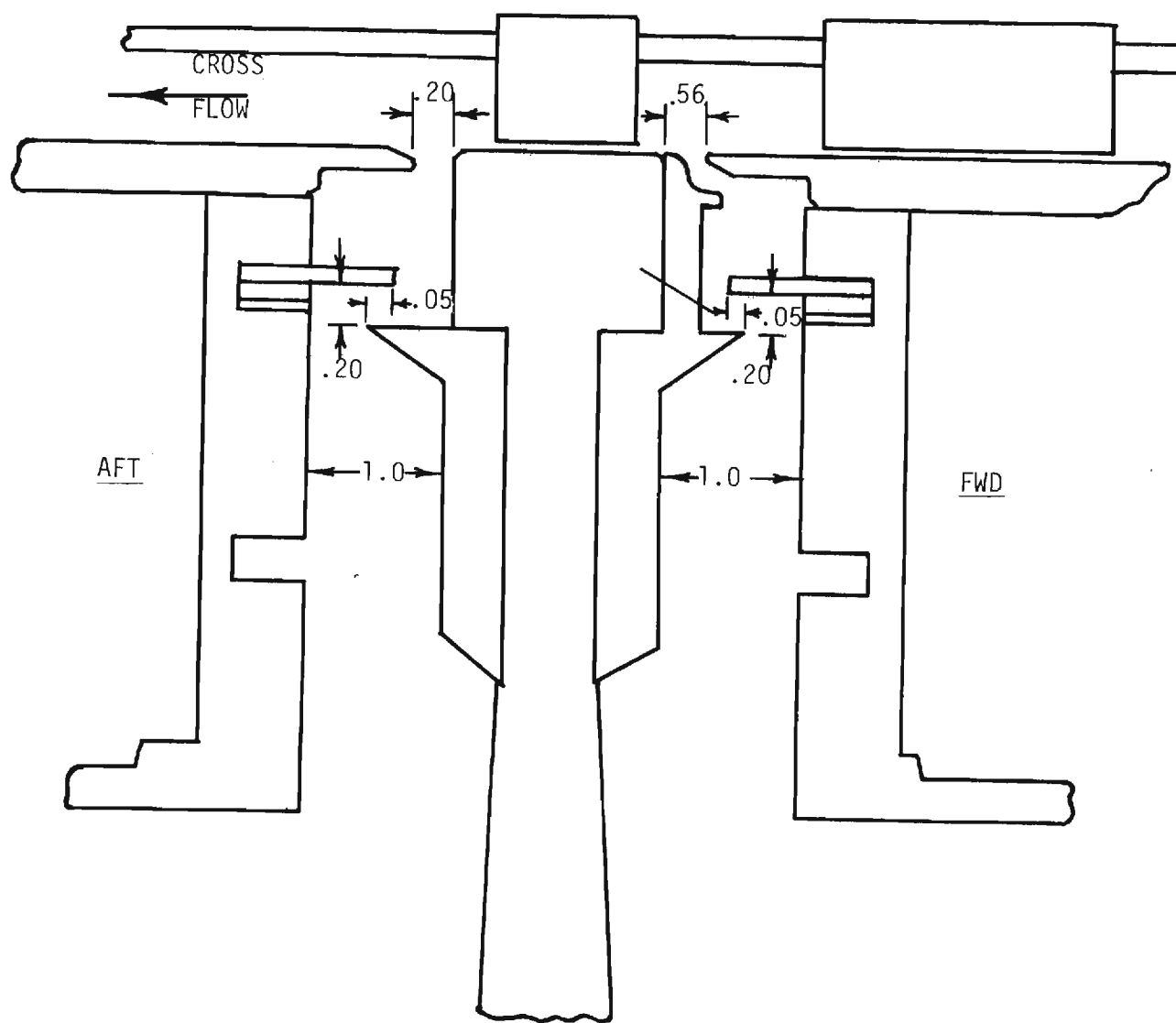


Figure 3. Baseline Configurations (Inches).

Table II. Summary of Tests Reported

Run No.	Conditions		Date Recorded ¹		Comments
	Cross flow temperature F	Wheel speed RPM	Pressure	Temperature	
B	250	2500	x	x	Varied fore and aft cooling flowrate
C	250	1000		x	Varied fore and aft cooling flowrate
D	250	2000		x	Varied fore and aft cooling flowrate
E	250	0	x	x	Varied fore and aft cooling flowrate
F	250	3000		x	Varied fore and aft cooling flowrate
G	250	3000	x	x	Varied fore and aft cooling flowrate
H	100	3000	x		Varied fore and aft cooling flowrate
I	100	2000	x		Varied fore and aft cooling flowrate
J	100	1000	x		Varied fore and aft cooling flowrate
K	100	3000	x		Partially blocked cooling flow inlets
L	250	3000	x	x	Partially blocked cooling flow inlets
M ²	100	0	x		Varied fore and aft cooling flowrates
M ₁ ²	100	2000	x		Varied fore and aft cooling flowrates
N ³	100	0	x		No forced cooling flow

¹in addition to flowrates.²sixteen circumferential pressure taps added.

IV. TEST RESULTS AND DISCUSSION OF RESULTS

The data from all tests listed in Table III are given in Appendix A. The experiments were preliminary in nature and fall into two broad categories - baseline confirmation of system performance and search for sources of asymmetry in the system. Tests B-G were run with hot cross flow while varying the cooling flow and wheel speed. It became apparent from these runs that the pressure measurements were possibly the more sensitive and faster measure than hot gas inflow. The temperature measurements were influenced by conduction through the structure which resulted in long starting transients. Several unheated tests (H-J) were then performed in which temperatures were not recorded. These experiments permitted faster exploration for the origins of seal flow variations.

It became apparent from the above experiments that the hot gas inflow to the wheel-space was asymmetrically distributed and that the instrumentation was inadequate for measuring it in any detail. On the belief that the circumferential variations were the result of variations in the mainflow, twelve additional circumferentially placed pressure taps were installed in the mainflow outer surface. Experiments M and N were then conducted which confirm that the circumferential variations of pressure exist in the mainflow and that additional temperature and pressure sensors are necessary before further experiments are conducted.

In two experiments (K,L) the wheel-space cooling flow inlets were varied to determine if they were the source of the asymmetry in the seal flow. There are three inlets to each wheel-space and one or two were blocked to determine the effect on seal flow. These experiments

Table III. Cooling Flowrates - lbm/sec
(Cross Flow - 4 lbm/sec for all tests)

RUN	FWD	AFT	COMMENTS	RUN	FWD	AFT	COMMENTS
B-1	-	-	INLET CLOSED	F-1	---	.38	FWD PLUGGED
B-2	.21	.16		F-2	.45	.28	
B-3	.25	.30		F-3	.40	.20	
B-4	.27	.38		F-4	.36	.10	
B-5	.25	.31		F-5	.32	0.0	
B-6	.225	.225		G-1	.618	.512	
B-7	.16	.113		G-2	.558	.443	
B-8	-	-	INLET CLOSED	G-3	.520	.381	
C-1	.28	-.16	INLET CLOSED	G-4	.464	.321	
C-2	.356	0.0		G-5	.381	.153	
C-3	.41	.178		G-6	.335	.554	
C-4	.46	.276	FULL INLET	G-7	.273	.362	
C-5	.46	.276	1500 RPM	H-1	.09	0	ONE FWD INLET
C-6	.406	.178	1500 RPM	H-2	.205	.205	ONE FWD INLET
C-7	.406	.178	COASTING	H-3	.275	.348	ONE FWD INLET
C-8	0	.225	FWD PLUGGED	H-4	.320	.52	ONE FWD INLET
C-9	0	.321	FWD PLUGGED	H-5	.35	.61	ONE FWD INLET
C-10	0	.414	FWD PLUGGED	H-6	.665	.57	
C-11	0	.474	FWD PLUGGED	J-1	.306	-.079	
C-12	0	0.0		J-2	.488	.316	
C-13	0	.149		J-3	.699	.587	
C-14	0	.105		J-4	.411	.158	
D-1	.47	.29		J-5	.775	.671	
D-2	.425	.21		K-1	.52	.40	ALL INLETS #1,3&
D-3	.386	.14		K-2	.69	.60	
D-4	.34	0		K-3	.40	.22	
D-5	.26	-.16		K-4	.55	.48	INLETS #1&3 FWD &
D-6	0	.36	FWD OFF	K-5	.46	.40	INLETS #1&3 FWD &
D-7	0	.46	FWD OFF	K-6	.31	.22	INLETS #1&3 FWD &
E-1	.47	.28		K-7	.31	.22	INLETS #3 FWD, 1 A
E-2	.42	.20		L-1	.42	.26	INLET #3 FWD, 1 AF
E-3	.39	.10		L-2	.40	.26	INLET #1&3 FWD & A
E-4	.37	.06		L-3	.53	.43	INLET #1&3 FWD & A
E-5	.36	0		L-4	.57	.43	ALL INLETS
E-6	0	.40	FWD OFF	L-5	.46	.26	ALL INLETS
E-11	.65	.53		M-1	.10	-.24	NO FORCED COOLING
E-12	.52	.37		M-2	.69	.59	MAX COOLING
E-13	.47	.28		M ₁ -1	.10	-.21	NO FORCED COOLING
E-14	.41	.18		M ₂ -2	.68	-.60	MAX COOLING
E-15	.39	.10		N			NO FORCED COOLING

were inconclusive.

From experiments B-F the general trend of behavior can be seen. Figure 4 shows the effect of wheelspeed and cooling air flow on the air temperature in the aft cooling space just inside the seals. It appears from this figure that the wheel speed does not have a very influential role.

Figure 5 shows the effect of cooling air flowrate on both the fore and aft air temperatures in the wheel space at 3000 RPM. In retrospect it is thought that the system had not yet reached thermal equilibrium when these data were taken. However, the expected trend of wheelspace temperature decrease with increasing cooling flow is clear.

Upon examination of the data from the above experiments it became apparent that for net cooling flowrates of up to 0.2 lb /s there was still some locations around the seal where there was hot gas flow into the wheelspace. This can be seen by studying the pressure differential across the seal at the two locations instrumented as a function of net cooling flow in the wheelspace. This is shown in Figure 5 for run H. Figure 6 shows that up to a net cooling flow of 0.2 lb/s there is still flow from the rim into the wheelspace at circumferential location 3.

As a result of the above observation attention was directed toward determining the source of the circumferential variation in seal flow (or seal pressure drop). Several possible sources were considered - all related to the geometry of the apparatus. Experiments K and L were to determine the effect of cooling air inlet ports. The

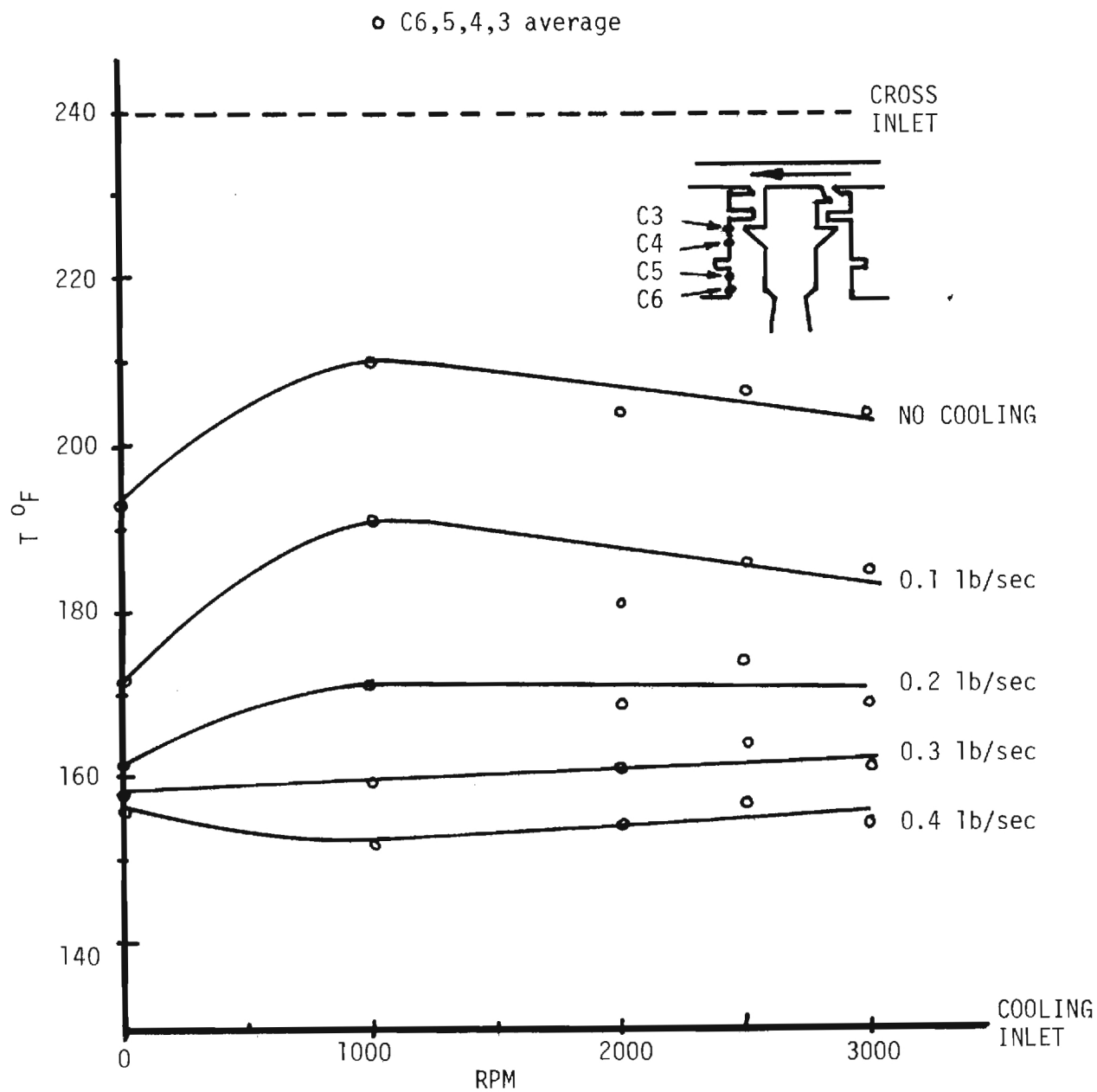


Figure 4 . Aft Temperature vs. RPM RUNS B-F.

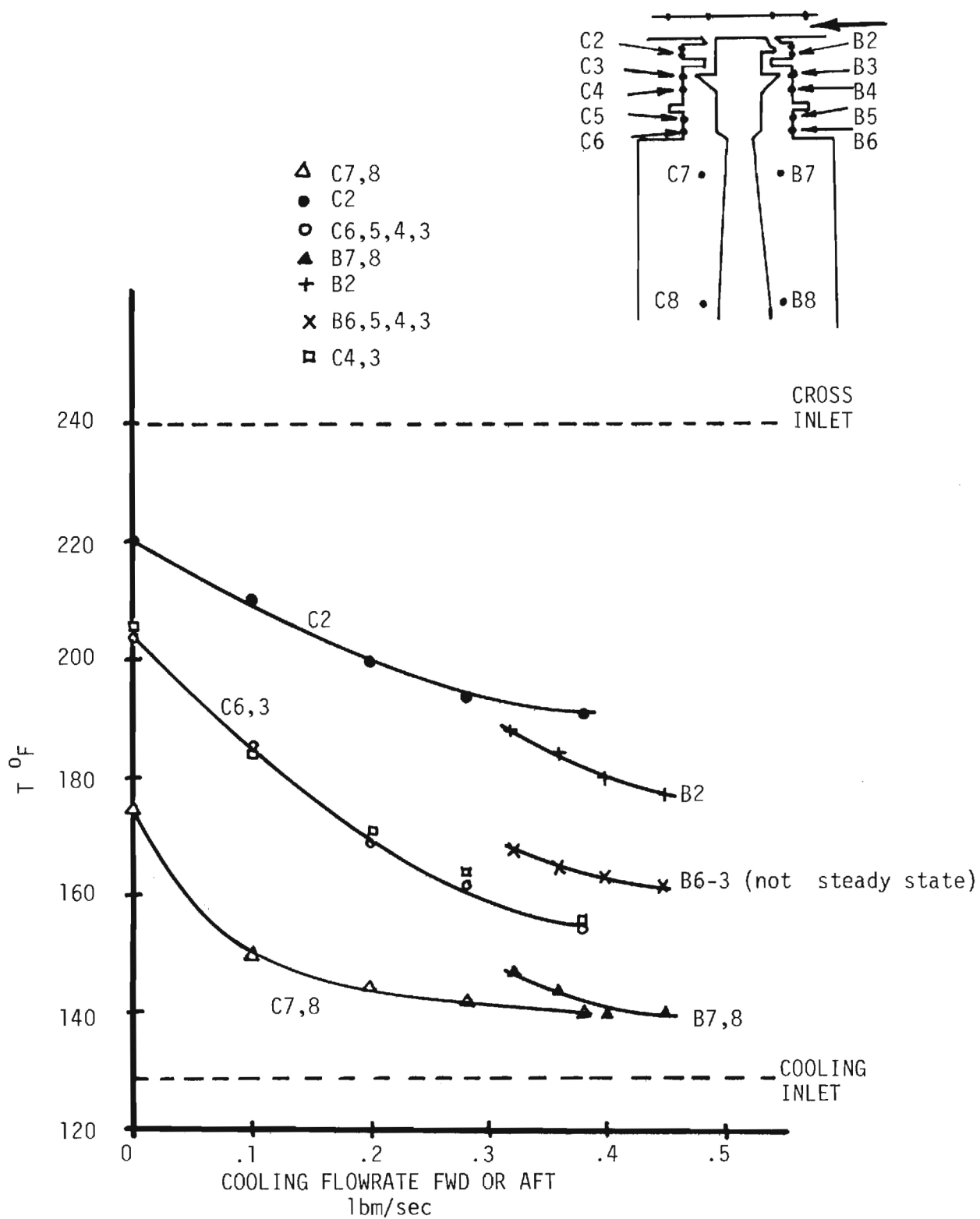
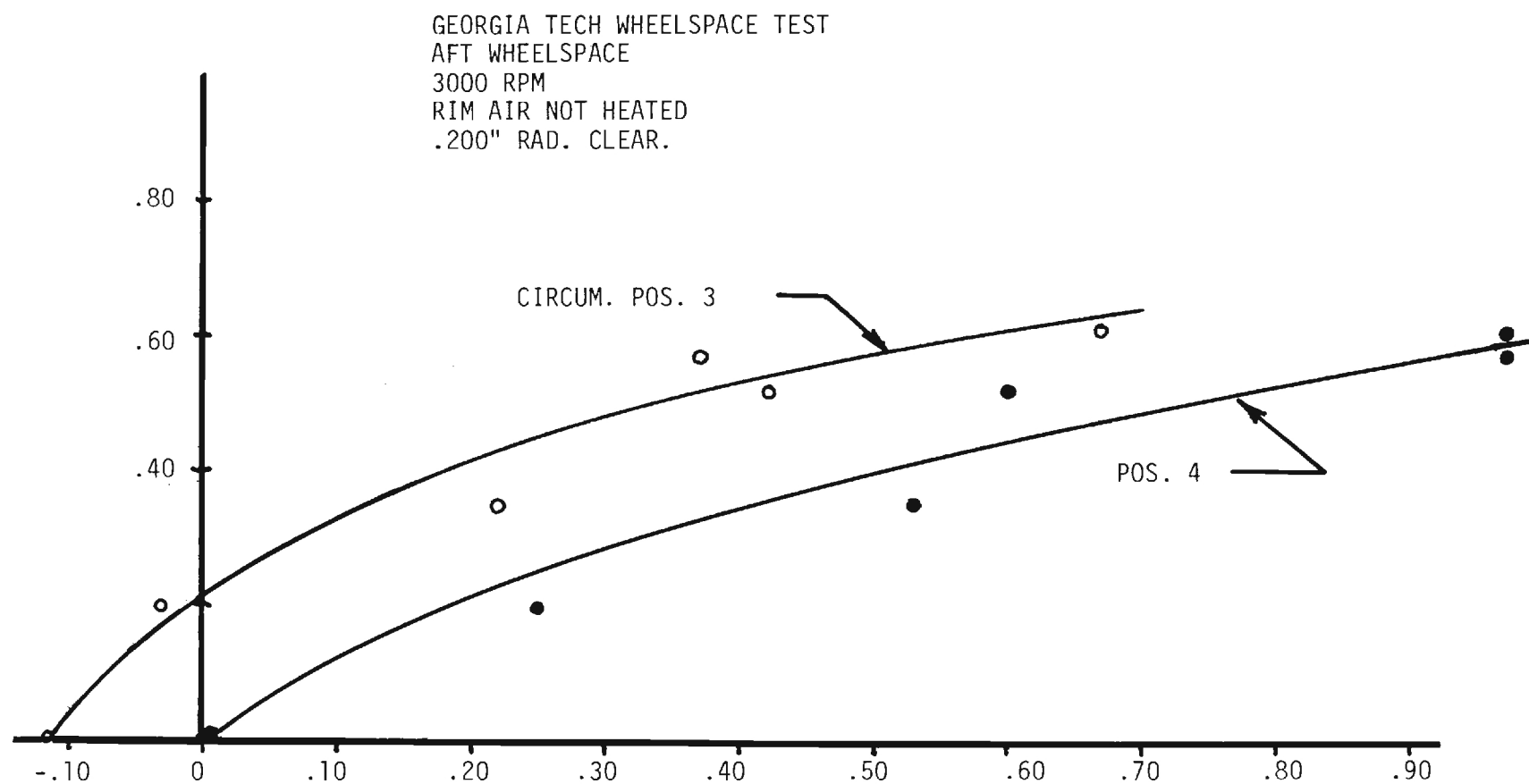


Figure 5 . Fwd and Aft Temperatures-3000 RPM.



results were negative. Next circumferential variation in seal geometry was considered but ruled out because it was thought that it would cause the seal pressure drops to vary with wheelspeed which appeared not to be the case. (However, the effect of seal geometry was not thoroughly tested and its role can not be totally excluded.) The circumferential variation of the rim flow was considered next. This was a likely source of the flow variations because of the discrete inlets and outlets for the rim flow.

To assess the circumferential variation of the rim flow a series of sixteen static pressure taps were added in pairs adjacent to the original tape (#3) in the outer casing. Half the taps were located over the forward seal and half over the aft seal. They are located relative to the blades and the rim flow inlets and outlets as shown in Figure 7. Later four more pairs were added to the right of #3.

Figure 7 also shows the magnitude of the static pressure at each tap by the length of an arrow. These data are for no rotation of the wheel and no forced wheelspace cooling air. The datum, and the fore and aft wheelspace pressures, are also shown on the left hand side of the figure. The circumferential variation of the rim flow pressure is clearly seen to be dependent on blade location and rim inlet flow port. In the case shown, on the forward side there are locations where the flow is into the wheelspace and other locations where the flow is out of the wheelspace. On the aft side for the conditions shown the seal pressure drop varies but the flow is always into the wheelspace.

Figure 8 shows the same type of data as discussed above but for two wheel speeds (0 and 2000 RPM) and for two wheelspace cooling flows

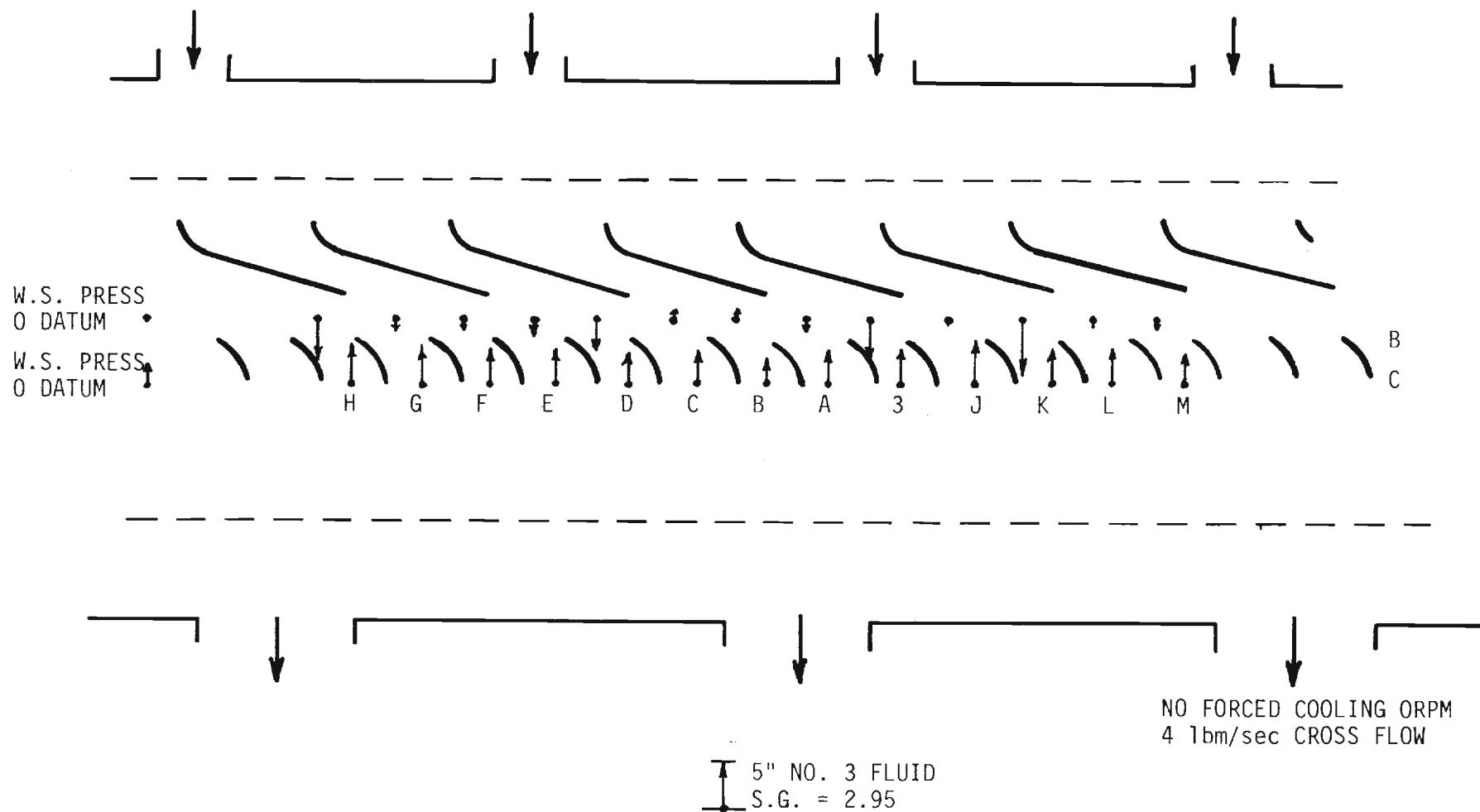


Figure 7. Cross Flow Pressures.

5" No. 3 Fluid (S.G. = 2)

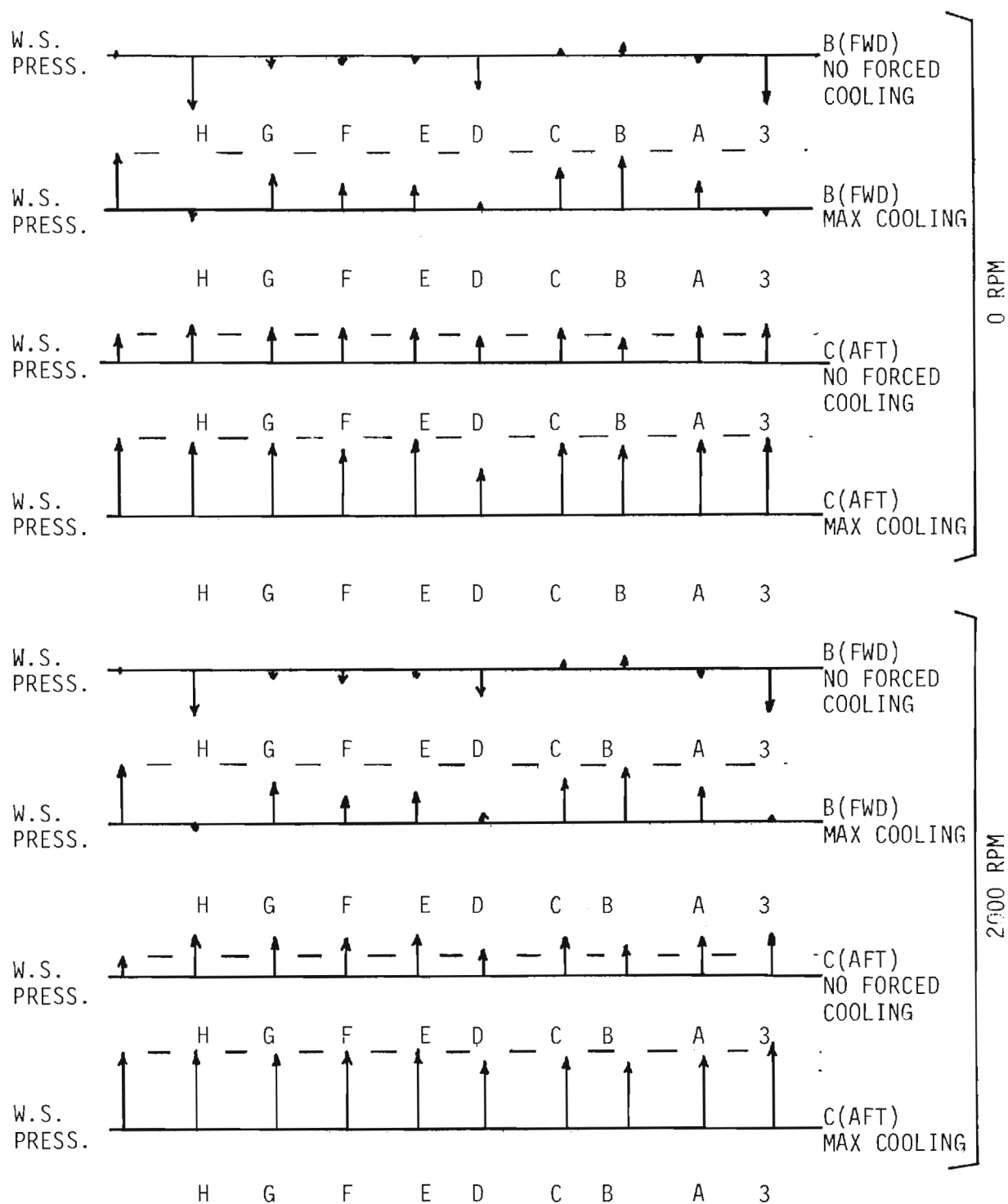


Figure 8. Cross Flow Pressures. (Run M)

(no forced cooling and maximum forced cooling). Again the average wheelspace pressure is shown on the left hand side for each case. In each case comparison of the local rim flow static pressure and the wheelspace pressure will indicate whether the flow is into the wheelspace or out of the wheelspace. Even at 2000 RPM and maximum cooling flow there exists both in and outward flow on the aft seal and on the forward seal while most flow is outward, at least one location appears to have approximately zero flow.

The above data show that the non-uniformity introduced in the rim flow by the inlet ports and the stator blades causes the circumferential variation of the seal pressure drop and flow. The possibility of hot gas inflow persists at high wheel speed even with substantial cooling air flow in the wheelspace.

V. CONCLUSIONS

The major conclusions of this phase of the wheelspace program are:

1. A test facility has been constructed and shown to operate which simulates all the salient features of an actual turbine wheel sealing system which include: hot rim flow, turning of rim flow, variable wheel speed, variable cooling flow. The facility is adequately instrumented to measure average flowrates, and local pressures and temperatures. The system seal geometry can be varied.
2. The system global behavior is as expected in an actual product turbine. The system detailed behavior - the circumferential variation of seal pressure drop - is assumed to simulate production machine behavior.
3. Preliminary baseline system behavior with respect to wheelspace temperature and pressure have been obtained and presented for different wheelspeeds and cooling flows.
4. Circumferential variation in main flow pressures is the dominate cause of hot gas flow into the wheelspace even at relatively high cooling flow rates. The circumferential variation in the main flow is the result of discrete inlet geometry and stator blading.
5. In the next phase of this program it will be necessary to record more circumferentially detailed pressure and temperature information in the seal area.

APPENDIX A

TABLES

Table A1. Temperatures

Table A2. Pressures

Table A1. Temperatures.

* Not Steady State

Table A2. Pressures (inches #3 Fluid - SP. GR. = 2.95).

ZONE	B-1	B-2	B-3	B-4	B-5	B-6	B-7	B-8	B-9	B-10	B-11	B-12	B-13	B-14	B-15	C-1	C-2	C-3	C-4	C-5	C-6	C-7	C-8	C-9	C-10	C-11	C-12	C-13	C-14	C-15	C-16	C-17	C-18	C-19	C-20	C-21	C-22	C-23	C-24	C-25	C-26	C-27	C-28	C-29	C-30	C-31	C-32	C-33	C-34	C-35	C-36	C-37	C-38	C-39	C-40	C-41	C-42	C-43	C-44	C-45	C-46	C-47	C-48	C-49	C-50	C-51	C-52	C-53	C-54	C-55	C-56	C-57	C-58	C-59	C-60	C-61	C-62	C-63	C-64	C-65	C-66	C-67	C-68	C-69	C-70	C-71	C-72	C-73	C-74	C-75	C-76	C-77	C-78	C-79	C-80	C-81	C-82	C-83	C-84	C-85	C-86	C-87	C-88	C-89	C-90	C-91	C-92	C-93	C-94	C-95	C-96	C-97	C-98	C-99	C-100	C-101	C-102	C-103	C-104	C-105	C-106	C-107	C-108	C-109	C-110	C-111	C-112	C-113	C-114	C-115	C-116	C-117	C-118	C-119	C-120	C-121	C-122	C-123	C-124	C-125	C-126	C-127	C-128	C-129	C-130	C-131	C-132	C-133	C-134	C-135	C-136	C-137	C-138	C-139	C-140	C-141	C-142	C-143	C-144	C-145	C-146	C-147	C-148	C-149	C-150	C-151	C-152	C-153	C-154	C-155	C-156	C-157	C-158	C-159	C-160	C-161	C-162	C-163	C-164	C-165	C-166	C-167	C-168	C-169	C-170	C-171	C-172	C-173	C-174	C-175	C-176	C-177	C-178	C-179	C-180	C-181	C-182	C-183	C-184	C-185	C-186	C-187	C-188	C-189	C-190	C-191	C-192	C-193	C-194	C-195	C-196	C-197	C-198	C-199	C-200	C-201	C-202	C-203	C-204	C-205	C-206	C-207	C-208	C-209	C-210	C-211	C-212	C-213	C-214	C-215	C-216	C-217	C-218	C-219	C-220	C-221	C-222	C-223	C-224	C-225	C-226	C-227	C-228	C-229	C-230	C-231	C-232	C-233	C-234	C-235	C-236	C-237	C-238	C-239	C-240	C-241	C-242	C-243	C-244	C-245	C-246	C-247	C-248	C-249	C-250	C-251	C-252	C-253	C-254	C-255	C-256	C-257	C-258	C-259	C-260	C-261	C-262	C-263	C-264	C-265	C-266	C-267	C-268	C-269	C-270	C-271	C-272	C-273	C-274	C-275	C-276	C-277	C-278	C-279	C-280	C-281	C-282	C-283	C-284	C-285	C-286	C-287	C-288	C-289	C-290	C-291	C-292	C-293	C-294	C-295	C-296	C-297	C-298	C-299	C-300	C-301	C-302	C-303	C-304	C-305	C-306	C-307	C-308	C-309	C-310	C-311	C-312	C-313	C-314	C-315	C-316	C-317	C-318	C-319	C-320	C-321	C-322	C-323	C-324	C-325	C-326	C-327	C-328	C-329	C-330	C-331	C-332	C-333	C-334	C-335	C-336	C-337	C-338	C-339	C-340	C-341	C-342	C-343	C-344	C-345	C-346	C-347	C-348	C-349	C-350	C-351	C-352	C-353	C-354	C-355	C-356	C-357	C-358	C-359	C-360	C-361	C-362	C-363	C-364	C-365	C-366	C-367	C-368	C-369	C-370	C-371	C-372	C-373	C-374	C-375	C-376	C-377	C-378	C-379	C-380	C-381	C-382	C-383	C-384	C-385	C-386	C-387	C-388	C-389	C-390	C-391	C-392	C-393	C-394	C-395	C-396	C-397	C-398	C-399	C-400	C-401	C-402	C-403	C-404	C-405	C-406	C-407	C-408	C-409	C-410	C-411	C-412	C-413	C-414	C-415	C-416	C-417	C-418	C-419	C-420	C-421	C-422	C-423	C-424	C-425	C-426	C-427	C-428	C-429	C-430	C-431	C-432	C-433	C-434	C-435	C-436	C-437	C-438	C-439	C-440	C-441	C-442	C-443	C-444	C-445	C-446	C-447	C-448	C-449	C-450	C-451	C-452	C-453	C-454	C-455	C-456	C-457	C-458	C-459	C-460	C-461	C-462	C-463	C-464	C-465	C-466	C-467	C-468	C-469	C-470	C-471	C-472	C-473	C-474	C-475	C-476	C-477	C-478	C-479	C-480	C-481	C-482	C-483	C-484	C-485	C-486	C-487	C-488	C-489	C-490	C-491	C-492	C-493	C-494	C-495	C-496	C-497	C-498	C-499	C-500	C-501	C-502	C-503	C-504	C-505	C-506	C-507	C-508	C-509	C-510	C-511	C-512	C-513	C-514	C-515	C-516	C-517	C-518	C-519	C-520	C-521	C-522	C-523	C-524	C-525	C-526	C-527	C-528	C-529	C-530	C-531	C-532	C-533	C-534	C-535	C-536	C-537	C-538	C-539	C-540	C-541	C-542	C-543	C-544	C-545	C-546	C-547	C-548	C-549	C-550	C-551	C-552	C-553	C-554	C-555	C-556	C-557	C-558	C-559	C-560	C-561	C-562	C-563	C-564	C-565	C-566	C-567	C-568	C-569	C-570	C-571	C-572	C-573	C-574	C-575	C-576	C-577	C-578	C-579	C-580	C-581	C-582	C-583	C-584	C-585	C-586	C-587	C-588	C-589	C-590	C-591	C-592	C-593	C-594	C-595	C-596	C-597	C-598	C-599	C-600	C-601	C-602	C-603	C-604	C-605	C-606	C-607	C-608	C-609	C-610	C-611	C-612	C-613	C-614	C-615	C-616	C-617	C-618	C-619	C-620	C-621	C-622	C-623	C-624	C-625	C-626	C-627	C-628	C-629	C-630	C-631	C-632	C-633	C-634	C-635	C-636	C-637	C-638	C-639	C-640	C-641	C-642	C-643	C-644	C-645	C-646	C-647	C-648	C-649	C-650	C-651	C-652	C-653	C-654	C-655	C-656	C-657	C-658	C-659	C-660	C-661	C-662	C-663	C-664	C-665	C-666	C-667	C-668	C-669	C-670	C-671	C-672	C-673	C-674	C-675	C-676	C-677	C-678	C-679	C-680	C-681	C-682	C-683	C-684	C-685	C-686	C-687	C-688	C-689	C-690	C-691	C-692	C-693	C-694	C-695	C-696	C-697	C-698	C-699	C-700	C-701	C-702	C-703	C-704	C-705	C-706	C-707	C-708	C-709	C-710	C-711	C-712	C-713	C-714	C-715	C-716	C-717	C-718	C-719	C-720	C-721	C-722	C-723	C-724	C-725	C-726	C-727	C-728	C-729	C-730	C-731	C-732	C-733	C-734	C-735	C-736	C-737	C-738	C-739	C-740	C-741	C-742	C-743	C-744	C-745	C-746	C-747	C-748	C-749	C-750	C-751	C-752	C-753	C-754	C-755	C-756	C-757	C-758	C-759	C-760	C-761	C-762	C-763	C-764	C-765	C-766	C-767	C-768	C-769	C-770	C-771	C-772	C-773	C-774	C-775	C-776	C-777	C-778	C-779	C-780	C-781	C-782	C-783	C-784	C-785	C-786	C-787	C-788	C-789	C-790	C-791	C-792	C-793	C-794	C-795	C-796	C-797	C-798	C-799	C-800	C-801	C-802	C-803	C-804	C-805	C-806	C-807	C-808	C-809	C-810	C-811	C-812	C-813	C-814	C-815	C-816	C-817	C-818	C-819	C-820	C-821	C-822	C-823	C-824	C-825	C-826	C-827	C-828	C-829	C-830	C-831	C-832	C-833	C-834	C-835	C-836	C-837	C-838	C-839	C-840	C-841	C-842	C-843	C-844	C-845	C-846	C-847	C-848	C-849	C-850	C-851	C-852	C-853	C-854	C-855	C-856	C-857	C-858	C-859	C-860	C-861	C-862	C-863	C-864	C-865	C-866	C-867	C-868	C-869	C-870	C-871	C-872	C-873	C-874	C-875	C-876	C-877	C-878	C-879	C-880	C-881	C-882	C-883	C-884	C-885	C-886	C-887	C-888	C-889	C-890	C-891	C-892	C-893	C-894	C-895	C-896	C-897	C-898	C-899	C-900	C-901	C-902	C-903	C-904	C-905	C-906	C-907	C-908	C-909	C-910	C-911	C-912	C-913	C-914	C-915	C-916	C-917	C-918	C-919	C-920	C-921	C-922	C-923	C-924	C-925	C-926	C-927	C-928	C-929	C-930	C-931	C-932	C-933	C-934	C-935	C-936	C-937	C-938	C-939	C-940	C-941	C-942	C-943	C-944	C-945	C-946	C-947	C-948	C-949	C-950	C-951	C-952	C-953	C-954	C-955	C-956	C-957	C-958	C-959	C-960	C-961	C-962	C-963	C-964	C-965	C-966	C-967	C-968	C-969	C-970	C-971	C-972	C-973	C-974	C-975	C-976	C-977	C-978	C-979	C-980	C-981	C-982	C-983	C-984	C-985	C-986	C-987	C-988	C-989	C-990	C-991	C-992	C-993	C-994	C-995	C-996	C-997	C-998	C-999	C-1000
COA																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																							

APPENDIX B

PHOTOGRAPHS OF WHEELSPACE APPARATUS

Figure B1. Overview of Wheelspace Test Facility.

Figure B2. Side View of Wheelspace Test Apparatus.

Figure B3. Inside View of Aft Wheelspace with Wheel Removed.

Figure B4. Rim Cover with Blades and Rim Flow Instrumentation.

Figure B5. Seal Area Instrumentation.

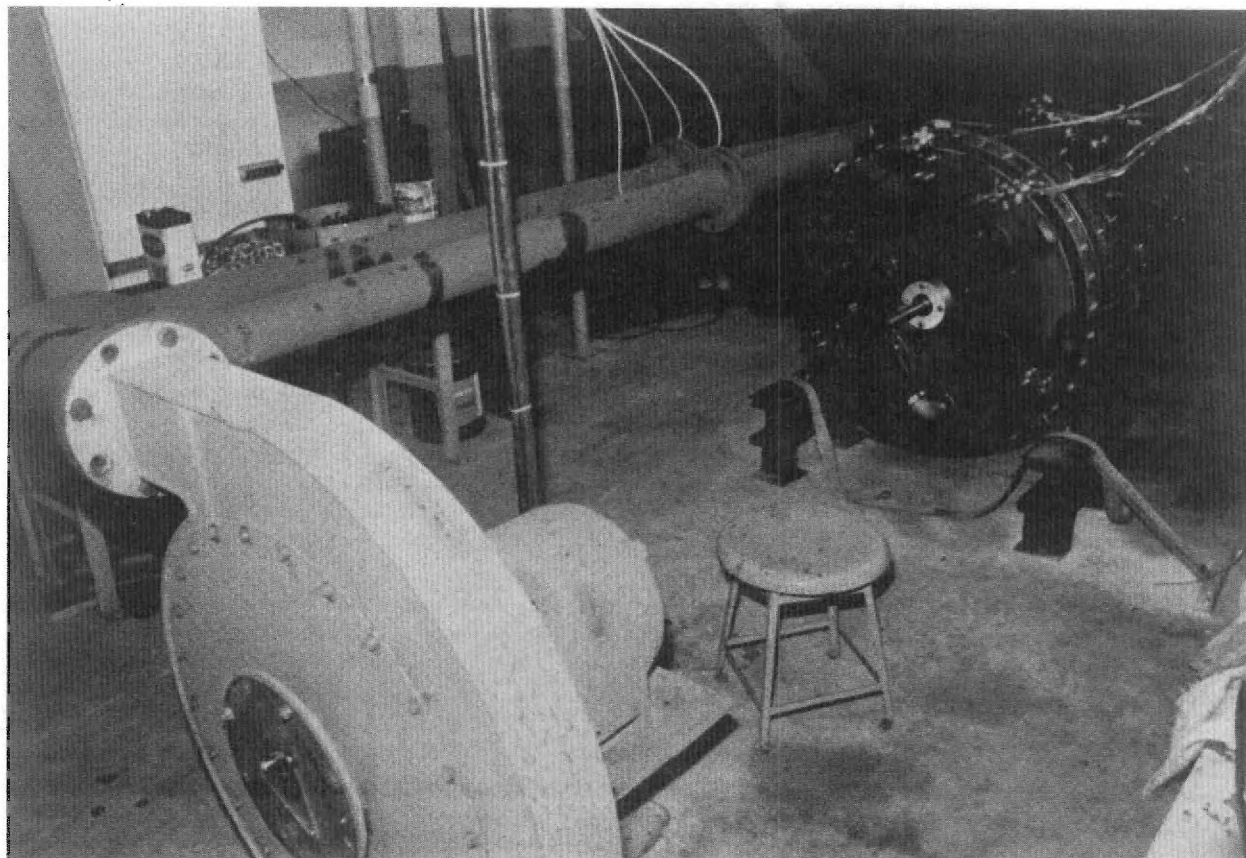


Figure B1. Overview of Wheelspace Test Facility. Wheelspace Apparature right background viewed from aft side, left foreground is blower to supply cooling air, drive engine behind wheelspace apparatus.

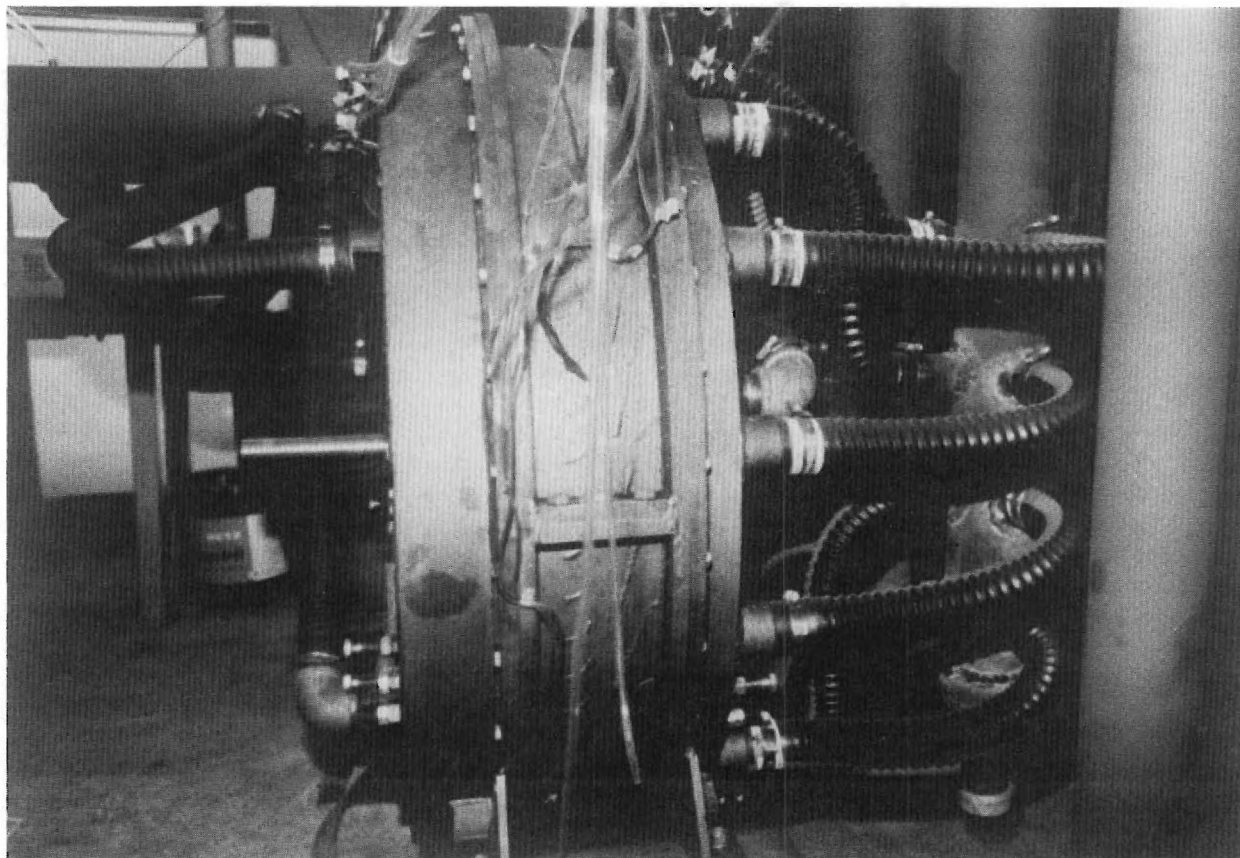


Figure B2. Side View of Wheelspace Test Apparatus. Rim flow enters on right and exits on left, forward wheelspace on right and aft on left, drive engine is on right.

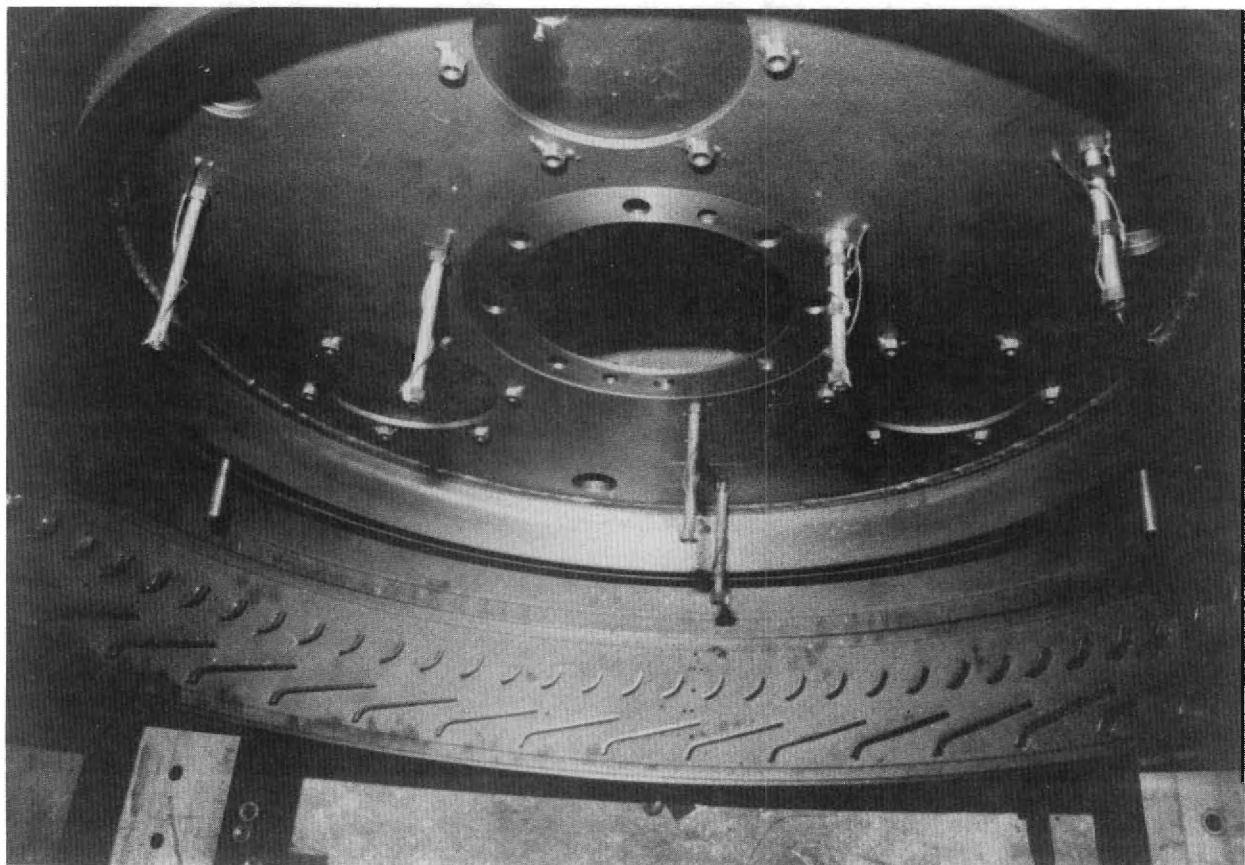


Figure B3. Inside View of Aft Wheelspace with Wheel Removed. Three cooling flow inlets (bottom center, top left and middle right), wheelspace thermocouples mounted threaded rod projecting from surface.

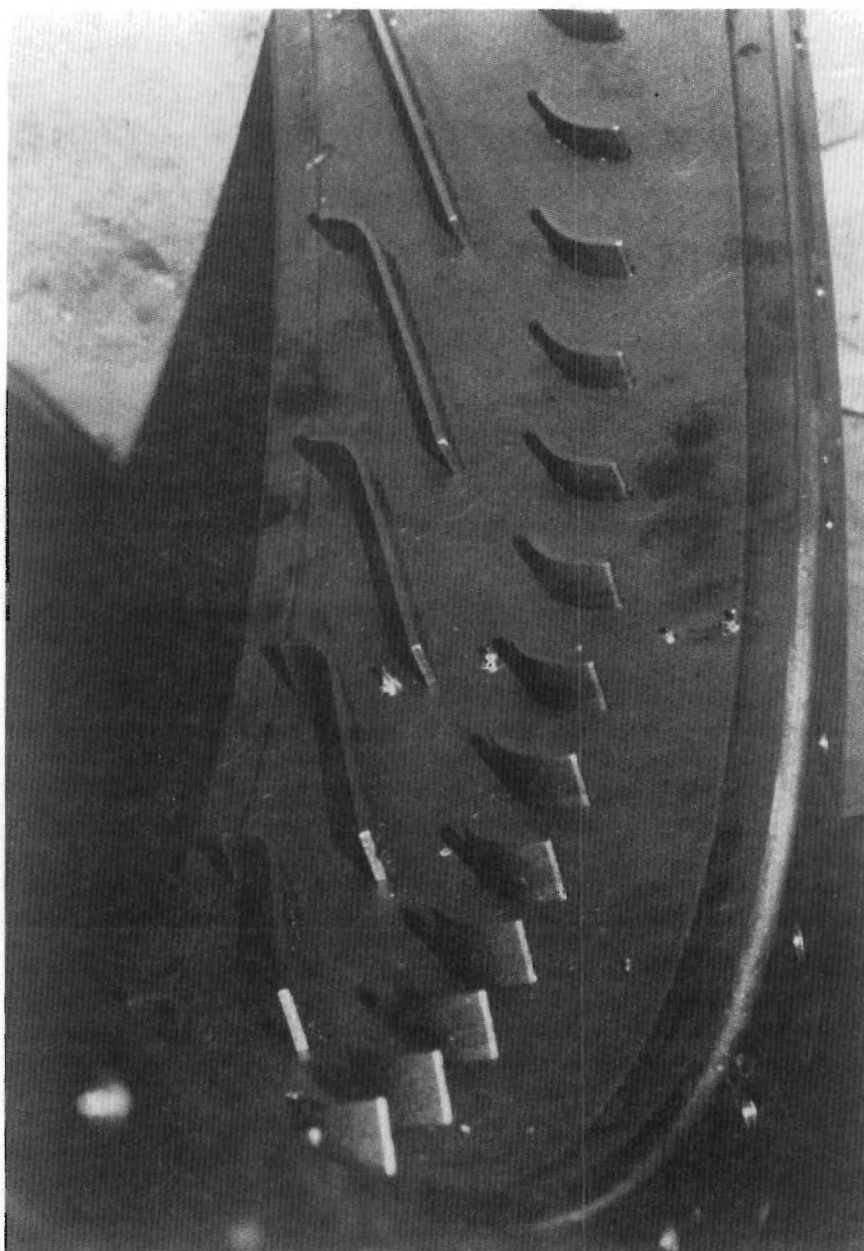


Figure B4. Rim Cover with Blades and Rim Flow Instrumentation. Nozzle blades on left and wheel turning blades on right, rim flow from left to right, rim flow instrumentation along a horizontal line in middle of picture.

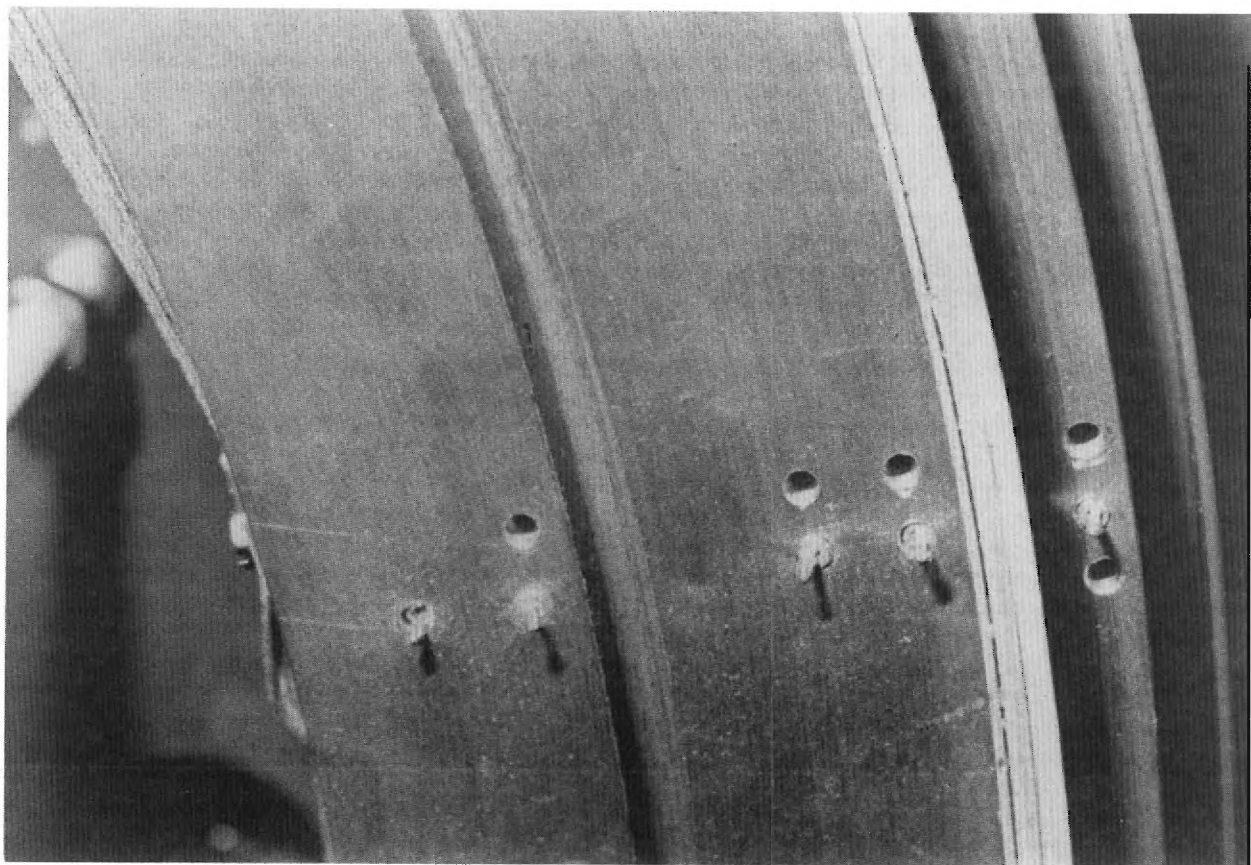
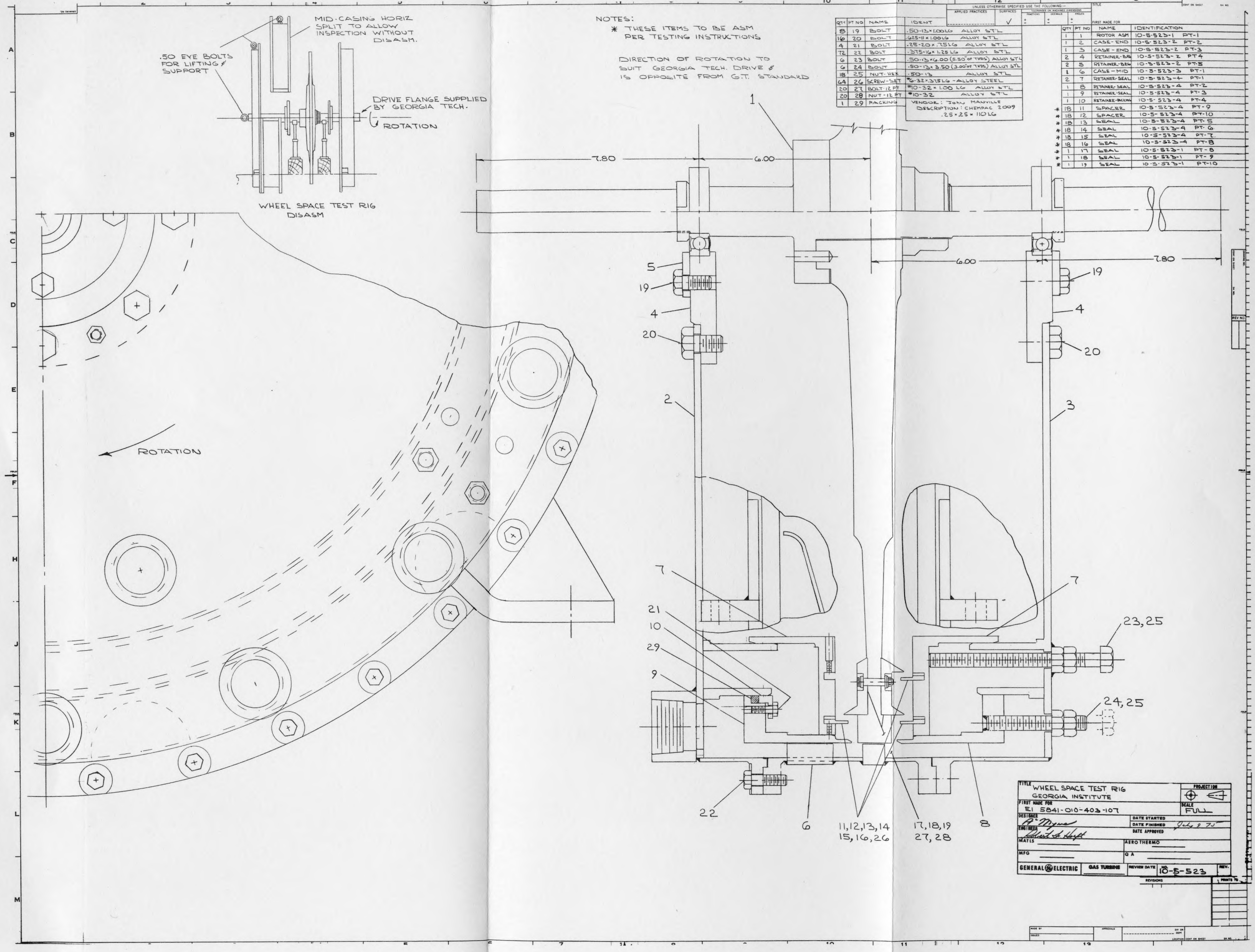


Figure B5. Seal Area Instrumentation.
Radically distributed pressure
taps and thermocouples in seal
area typical of #3 circumferential
position (forward or aft).

APPENDIX C

MECHANICAL DRAWINGS OF WHEELSPACE APPARATUS

No. 10-5-523	Wheelspace Test Rig
No. 10-5-523-1	Wheelspace Test Rig
No. 10-5-523-2	Wheelspace Test Rig
No. 10-5-523-3	Wheelspace Test Rig
No. 10-5-523-4	Wheelspace Test Rig



NOTES:
* THESE ITEMS TO BE ASM
PER TESTING INSTRUCTIONS

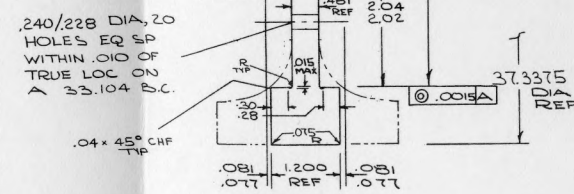
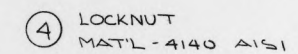
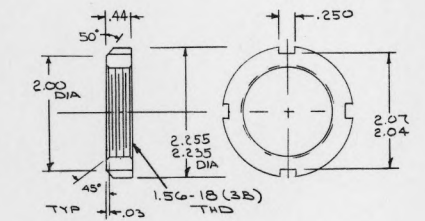
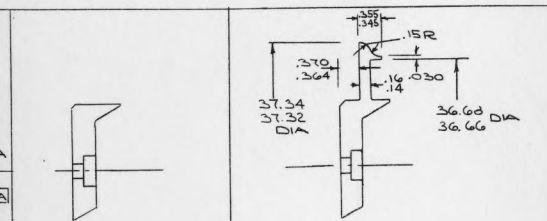
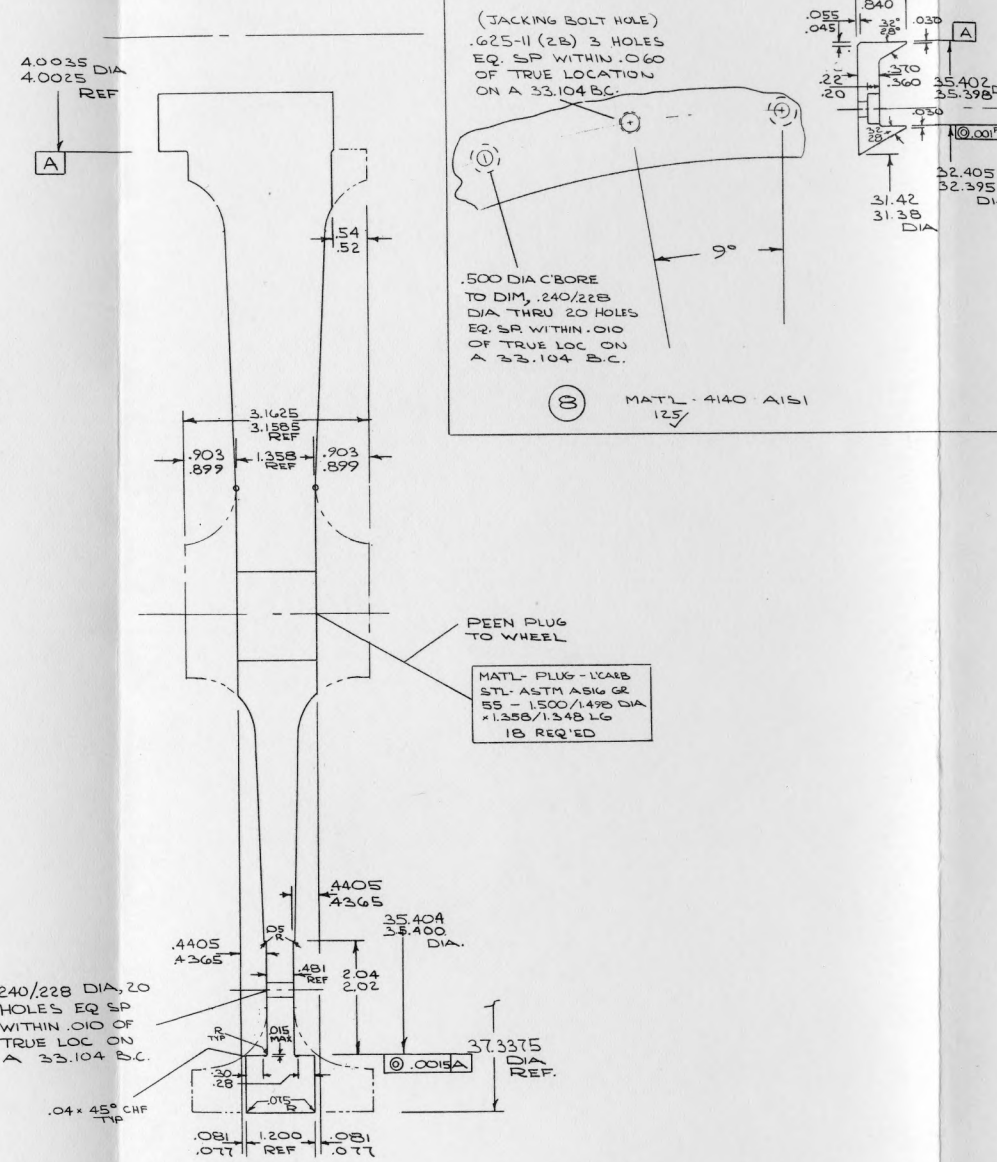
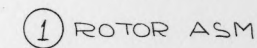
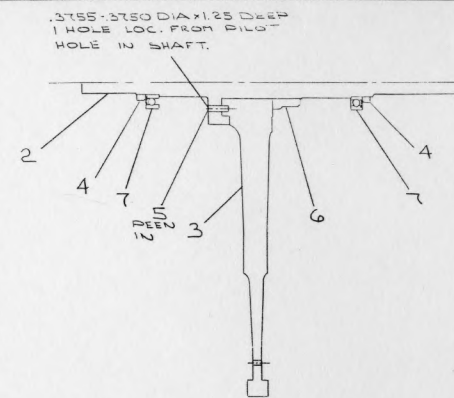
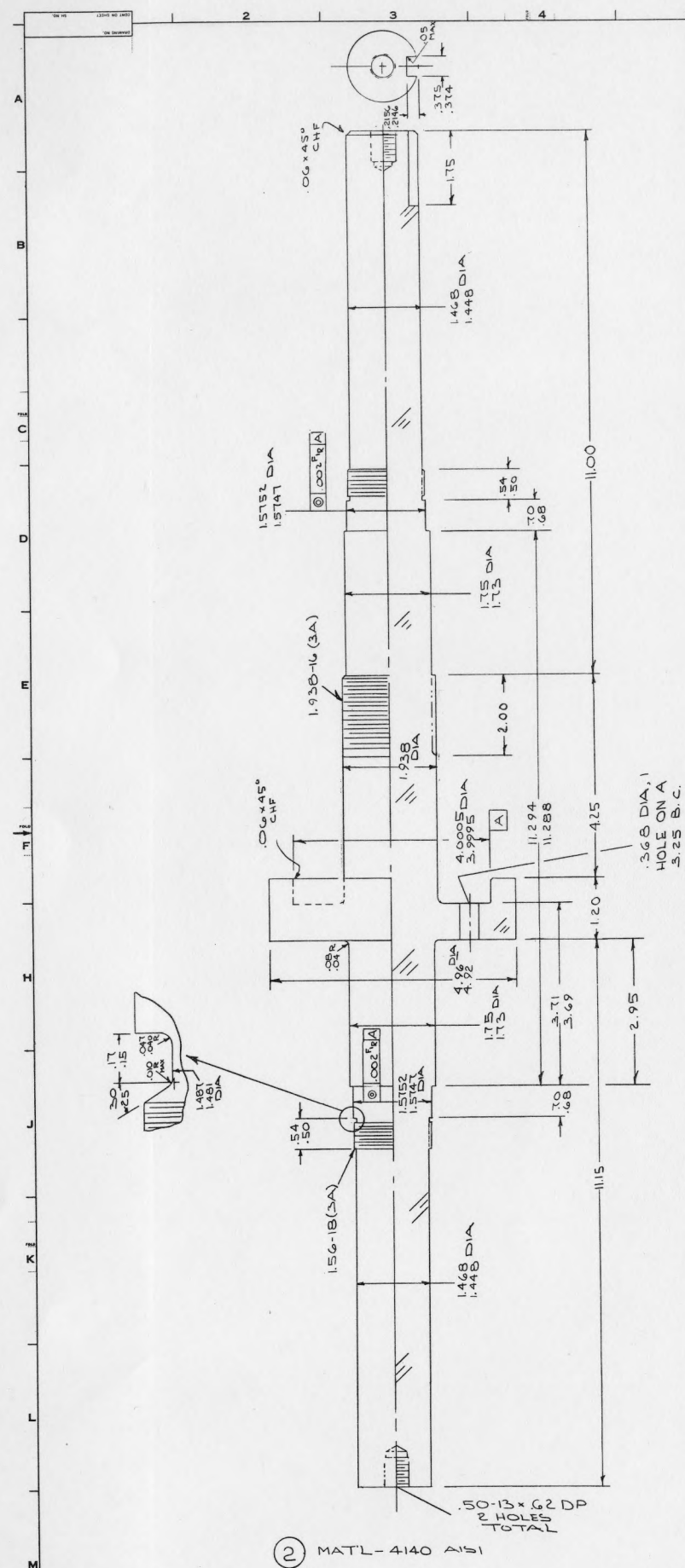
DIRECTION OF ROTATION TO
SUIT GEORGIA TECH. DRIVE &
IS OPPOSITE FROM G.T. STANDARD

QTY	PT NO	NAME	IDENT	APPLIED PRACTICES	UNLESS OTHERWISE SPECIFIED USE THE FOLLOWING:	REVISIONS
1	19	BOLT	.50-13X1.000 LG ALLOY STL			
1	20	BOLT	.415-11X1.000 LG ALLOY STL			
1	21	BOLT	.25-20X1.125 LG ALLOY STL			
1	22	BOLT	.275-16X1.125 LG ALLOY STL			
1	23	BOLT	.50-13X1.000 LG (5.50" TYP) ALLOY STL			
1	24	BOLT	.50-13X1.350 (3.00" TYP) ALLOY STL			
1	25	NUT-1/2"	.50-13 ALLOY STL			
1	26	SCREW-SET	6-32X.375 LG - ALLOY STEEL			
1	27	BOLT-1/2"	10-32X1.00 LG ALLOY STL			
1	28	NUT-1/2"	10-32 ALLOY STL			
1	29	PACKING				


VENDOR: JENNY MANVILLE
DESCRIPTION: CHERPAC 2009
.25X.25X1.10 LG

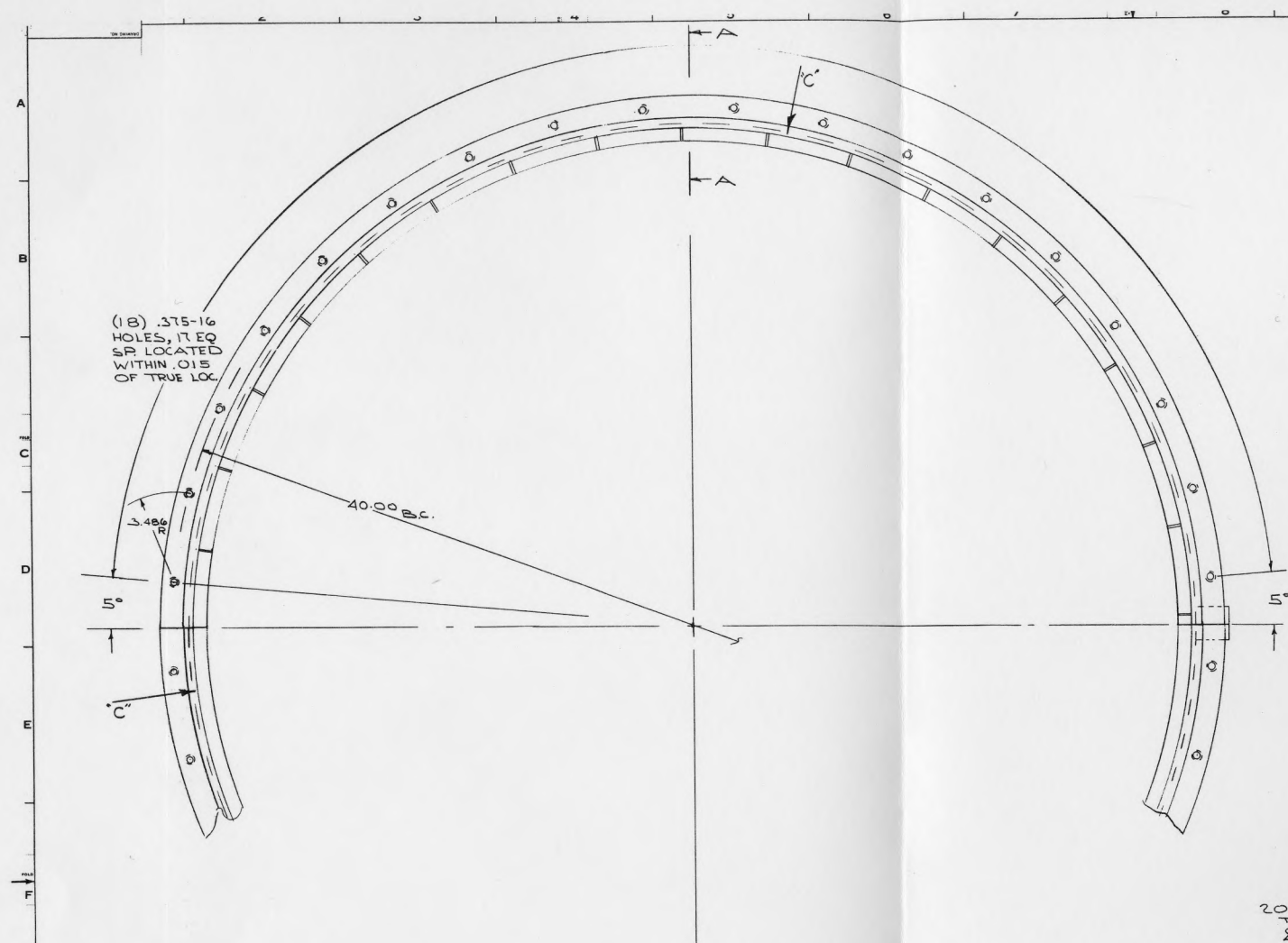
QTY	PT NO	NAME	IDENTIFICATION
1	1	ROTOR ASM	10-5-523-1 PT-1
1	2	CASE-END	10-5-523-2 PT-2
1	3	CASE-END	10-5-523-2 PT-3
2	4	RETAINER-BW	10-5-523-2 PT-4
2	5	RETAINER-BW	10-5-523-2 PT-5
1	6	CASE-MID	10-5-523-3 PT-1
2	7	RETAINER-SEAL	10-5-523-4 PT-1
1	8	RETAINER-SEAL	10-5-523-4 PT-2
1	9	RETAINER-SEAL	10-5-523-4 PT-3
1	10	RETAINER-SEAL	10-5-523-4 PT-4
1	11	SPACER	10-5-523-4 PT-9
1	12	SPACER	10-5-523-4 PT-10
1	13	SEAL	10-5-523-4 PT-5
1	14	SEAL	10-5-523-4 PT-6
1	15	SEAL	10-5-523-4 PT-7
1	16	SEAL	10-5-523-4 PT-8
1	17	SEAL	10-5-523-1 PT-6
1	18	SEAL	10-5-523-1 PT-9
1	19	SEAL	10-5-523-1 PT-10

TITLE WHEEL SPACE TEST RIG GEORGIA INSTITUTE		PROJECTION FIRST ANGLE	
FIRST MADE FOR E1 5841-010-403-107		SCALE FULL	
DESIGNED P. Myers	DATE STARTED July 2 75	DATE FINISHED July 2 75	
ENGINEER Robert A. Hight	DATE APPROVED		
MATERIALS Q A		AERO THERMO	
INFO		Q A	
GENERAL ELECTRIC		GAS TURBINE	
REVISION DATE 10-5-523		REV.	

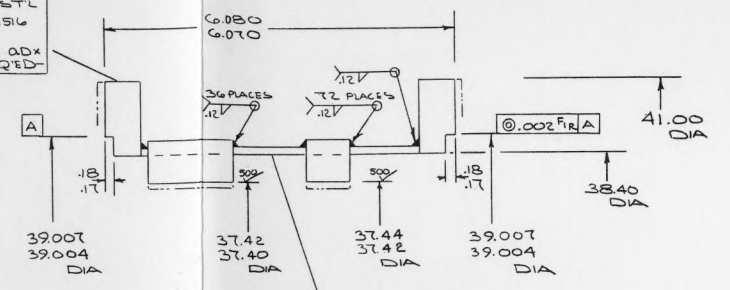


UNLESS OTHERWISE SPECIFIED
ALL MACH SURFACES 125/
(3) MAKE FROM 127D8303 PT-12
(WHEEL MATL - MOD. 4340)

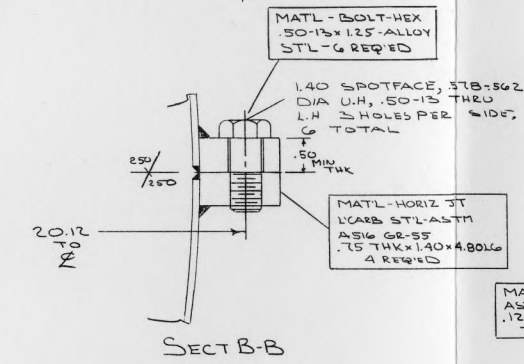
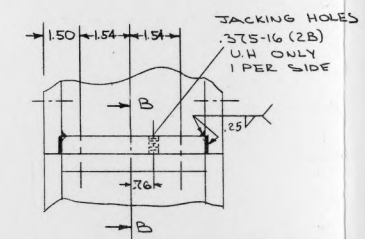
TITLE WHEEL SPACE TEST RIG		PROJECTION 	
FIRST NAME FOR E- 5841-010-403-107		SCALE	
DESIGNED BY <i>R. Myers</i>	DATE STARTED	DATE FINISHED	
ENGINEER <i>Robert A. Light</i>	DATE APPROVED		
MATERIALS	AERO THERMO		
INFO	Q A		
GENERAL ELECTRIC	GAS TURBINE	REVIEW DATE	NO. 10-5-523-1 REV.



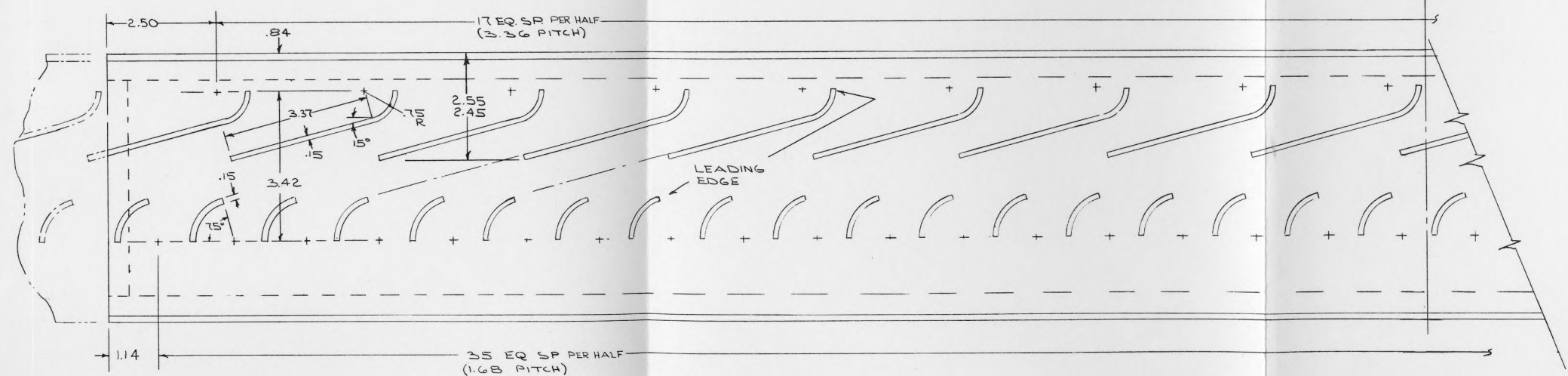
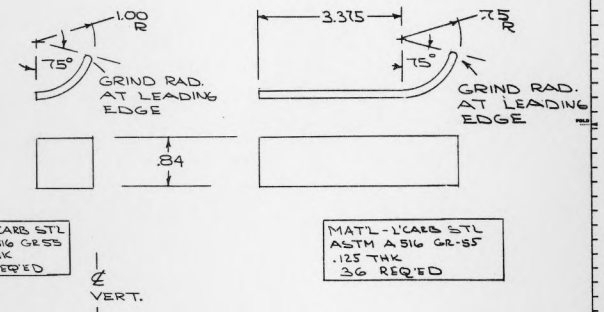
MATL-L'CARB STL
1/2 RING ASTM A516
GRADE 55
.125 THK x 4.10 O.D. x
38.40 I.D. - 4 REQ'D



SECTION A-A



SECT B-B



VIEW C-C

FULL SIZE DEVELOPMENT OF ONE QUADRANT
PITCH DIM'S TAKEN AT MEAN RAD.
OF OUTER WALL (19.25R)
ALL SLOTS TO BE LOCATED WITHIN
.25 OF TRUE LOCATION, PHYSICAL SIZE
SLOTS TO SUIT MFG METHODS

TITLE WHEELSPACE TEST RIG		PROJECTION FIRST ANGLE
FIRST MADE FOR E1-5841-010-403-107		SCALE 1:1
DESIGNER J. J. Jones	DATE STARTED 9-17-73	DATE FINISHED 9-17-73
ENGINEER J. J. Jones	DATE APPROVED 9-17-73	DATE APPROVED 9-17-73
MATERIALS G A	Q A	Q A
MFG GENERAL ELECTRIC	GAS TURBINE	REVIEW DATE 10-5-73

REVISIONS	PRINTS TO
1	1
2	2
3	3
4	4
5	5
6	6
7	7
8	8
9	9
10	10

